

## Solution for TSRT09 Control Theory, 2021-06-08

1. (a) Non-minimum phase systems have a lower phase margin than their minimum phase counterparts (hence they are more stability-critical), and cannot be feedback regulated arbitrarily fast, i.e., the closed-loop bandwidth must be upper bounded.
- (b) The least common denominator of all minors is  $p(s) = (s + 1)(s + 3)(s + 7)$ , hence the poles are  $s = -1$ ,  $s = -3$  and  $s = -7$ .
- (c) If  $v$  is a white noise of variance 1, then  $\Phi_v(\omega) = 1$ , hence when

$$y(s) = H(s)v(s) = \frac{1}{s + 2}v(s)$$

it is

$$\Phi_y(\omega) = |H(i\omega)|^2\Phi_v(\omega) = H(i\omega)H(-i\omega) \cdot 1 = \frac{1}{\omega^2 + 4}$$

- (d) When e.g.  $d = 3$ , then for  $s \rightarrow \infty$  it is  $y = u + 2$  meaning that no  $u$  s.t.  $|u| \leq 1$  can completely cancel the effect of the disturbance.
  - (e) For a physical system it is reasonable to assume that both plant and controller decay at least as fast as  $1/\omega$ , hence the Bode integral theorem can be applied. When the area A is bigger than the area B, then this theorem says that the loop gain must have at least one unstable pole.
2. (a) The pole polynomial is  $p(s) = (s + 1)^2(s - 2)$ , and the maximal minor (with  $p(s)$  as denominator) is

$$\det G(s) = \frac{3(s + 2 - \alpha)}{p(s)}$$

from which we see that the zero is  $z = -2 + \alpha$ . When  $\alpha < 2$  the system is minimum phase, and when  $\alpha > 2$  it is non-minimum phase.

- (b) Exact dynamical decoupling requires to cancel the zero in  $z = -2 + \alpha$ . However, when  $\alpha > 2$  then this requires to use  $F(s)$  with an unstable pole, which is a bad idea (Never use unstable regulators!).
- (c) Steady-state decoupling corresponds to choosing a constant  $F$  such that  $G(0)F$  is diagonal. For instance if  $F = G^{-1}(0)$  then  $\tilde{G}(0)$  is the identity matrix. Doing the calculations:

$$F = \frac{1}{2 - \alpha} \begin{bmatrix} 2 & -1 \\ -2\alpha/3 & 2/3 \end{bmatrix}$$

Notice that the term  $2 - \alpha$  appearing in the denominator is ill-posed only in the case  $\alpha = 2$  (rather than  $\alpha \leq 2$ , as it was in (b)).

3. (a) Since the penalty is quadratic we can use an LQ design to get  $u = -Lx$  where  $L$  is computed with the `lqr` function in matlab:

$$L = \begin{bmatrix} 1.7321 & 4.1492 & 4.9697 & 3.1527 \end{bmatrix}$$

The poles are in this case

```
eig(A-B*L) =
-0.5963 + 0.9801i
-0.5963 - 0.9801i
-0.9801 + 0.5963i
-0.9801 - 0.5963i
```

- (b) The output equation for the system is

$$y = Cx = \begin{bmatrix} 1 & 0 & 0 & 0 \\ 0 & 0 & 1 & 0 \end{bmatrix}$$

and the Kalman filter is

$$\dot{\hat{x}} = A\hat{x} + Bu + K(y - C\hat{x})$$

which, in order to get the full regulator, must be coupled with  $u = -L\hat{x}$ , where  $L$  is the one given above. The computation of  $K$  can be done using the `lqe` function of matlab (for each value of  $\rho$ ) and the corresponding poles of the Kalman filter are the eigenvalues of  $A - KC$ . For  $\rho = 10$  it is

$$K = \begin{bmatrix} 2.5680 & 0.1970 \\ 3.3166 & 0.9939 \\ 0.1970 & 2.5071 \\ 0.0058 & 3.1623 \end{bmatrix}$$

which corresponds to

```
eig(A-K*C) =
-1.1714 + 1.3661i
-1.1714 - 1.3661i
-1.3661 + 1.1714i
-1.3661 - 1.1714i
```

All real parts are more negative than those in (a) hence the poles of the Kalman filter are faster than those of the LQ regulator, which is normally a good rule of thumb.

- (c) The LQ regulator in (a) is robust (e.g. infinite amplitude margin) while the LQG in (b) is not, see Sect. 9.4 in the book.

4. (a) When  $u = 0$  the equilibria for the system are

$$x_1^* = \begin{bmatrix} 0 \\ 0 \end{bmatrix} \quad \text{and} \quad x_2^* = \begin{bmatrix} 1 \\ -1 \end{bmatrix}$$

The Jacobian linearization for the system is

$$A = \begin{bmatrix} x_2 & -1 + x_1 \\ 2 + 3x_2 & -1 + 3x_1 \end{bmatrix}, \quad B = \begin{bmatrix} 1 & 0 \\ 0 & 1 \end{bmatrix}$$

Computed at  $x_i^*$  it leads to

$$A_1 = \begin{bmatrix} 0 & -1 \\ 2 & -1 \end{bmatrix} \Rightarrow \text{eig}(A_1) = \{-0.5 \pm 1.3229i\} \Rightarrow x_1^* \text{ is an as. stable focus}$$

$$A_2 = \begin{bmatrix} -1 & 0 \\ -1 & 2 \end{bmatrix} \Rightarrow \text{eig}(A_2) = \{-1, 2\} \Rightarrow x_2^* \text{ is a saddle point}$$

- (b) The linearized system at  $x_2^*$  is  $(A_2, B, C)$  (with  $C = I$ ). Since it is controllable, pole placement can be done arbitrarily. For instance using an LQR design with  $Q_1 = Q_2 = I$ , one gets

$$L = \begin{bmatrix} 0.7305 & -1.0730 \\ -1.0730 & 3.9618 \end{bmatrix} \Rightarrow \text{eig}(A_2 - BL) = \{-1.5434, -2.1490\}$$

- (c) The linearized system for the stable equilibrium is  $(A_1, B, C)$ . Compute  $G(s) = C(sI - A_1)^{-1}B$  and observe that  $\text{RGA}(G(0)) = \begin{bmatrix} 0 & 1 \\ 1 & 0 \end{bmatrix}$ . Hence the pairing is  $u_1 \leftrightarrow y_2$  and  $u_2 \leftrightarrow y_1$ .

5. (a) The describing function  $Y_f(C)$  for the saturation is given in Example 14.5 of the book, and it is always real (and negative), see blue line in Fig. 1. Since  $r = 0$ , the linear part of the system can be written as

$$\tilde{G}(S) = G(S)F(S) = \frac{K(\tau s + 1)}{\tau s(s + 2)^2}$$

whose Nyquist curve is given in Fig. 1 for 2 values of  $K$  (red and green curves). When  $K$  grows there is an intersection between  $-1/Y_f(C)$  and  $\tilde{G}(i\omega)$ . Numerically, the intersection starts to appear when  $K \sim 2.67$ , and keeps being present when  $K$  grows, meaning that the prediction of the describing function method is that self-sustained oscillations are present for  $K > 2.67$ .

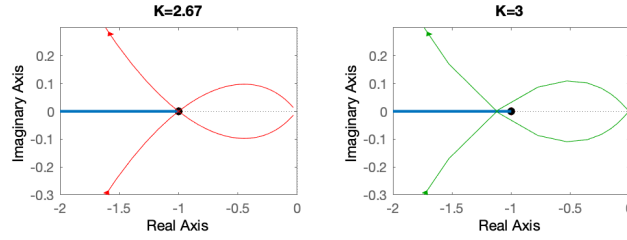


Figure 1: Describing function  $-\frac{1}{Y_f(C)}$  and Nyquist curve of the loop gain  $\tilde{G}(i\omega)$  for two values of  $K$ .

- (b) The frequency of the oscillations can be obtained by computing the value of  $\omega$  for which  $\tilde{G}(i\omega)$  is real (since  $-1/Y_f(C)$  is real). Doing the calculations:

$$\tilde{G}(i\omega) = \frac{10K(i\omega/10 + 1)}{i\omega(i\omega + 2)^2} = \frac{10K((-4\omega + 0.1\omega(4 - \omega^2)) - i(4 - 0.6\omega^2))}{\omega(16\omega^2 - (4 - \omega)^2)}$$

This expression is real when the imaginary term vanishes, i.e., for  $\omega = \sqrt{20/3} = 2.582$  (note that the fact that this value of  $\omega$  is similar to the value of  $K$  at which oscillations start is just a coincidence). The frequency does not change with  $K$ .

- (c) Since  $-1/Y_f(C)$  grows by moving from right to left, as in Fig. 14.9(a) of the book, the oscillations are stable.
- (d) Simulating the system (from nonzero initial conditions), stable self-sustaining oscillations can indeed be seen, see Fig. 2. Notice that oscillations seem to start for slightly slower values of  $K$  than those predicted by the describing function method (which is an approximation), although as  $t \rightarrow \infty$  they probably have  $C \rightarrow 0$  when  $K < 2.67$ .

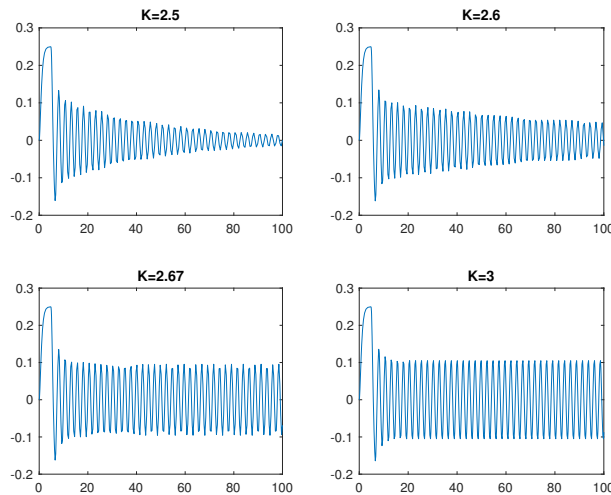


Figure 2: Oscillations as  $K$  is varied.