

**Solution for the “Modeling and Simulation” exam (TSRT62) 2018-08-27**

1. (a) A step response can give information for instance on the time constants of the system, on the sampling time, on the presence of oscillations, on its linear/nonlinear behavior, etc.
- (b) The first order implicit Adams method is nothing else than the Euler backwards method. On the test function  $\dot{x} = \lambda x$ , it becomes

$$x_{n+1} = x_n + h\lambda x_{n+1}$$

i.e.,

$$x_{n+1} = \frac{1}{1 - h\lambda} x_n$$

The stability region is  $|1 - h\lambda| > 1$ , i.e.,  $h\lambda$  must be outside a disk of radius 1 centered at 1. When  $\lambda < 0$  this is true for all  $h > 0$ . For us it is  $\lambda = -3$ , hence the Euler backwards discretization of our system  $\dot{x} = -3x$  is stable for all  $h > 0$ .

- (c) Stiff systems are difficult because fast and slow modes coexist. One simple way to quantify stiffness of a (linear) system is to look at its eigenvalues. The eigenvalues of the 3 systems are  $\{-1, -2\}$ ,  $\{-0.001, -1\}$ , and  $\{-1, -100\}$ . The ratio between them is 2, 1000 and 100. Hence (i) is the least stiff system, followed by (iii), while (ii) is the stiffest.
- (d) Modelica is an object-oriented language. When you connect blocks constructed in this language, some of the variables may become algebraic equations (describing the block interconnections), rather than differential equations.

2. Asymptotically it is

$$\begin{aligned} V(b) &= \lim_{N \rightarrow \infty} \frac{1}{N} \sum_{t=1}^N [\hat{y}(t) - y(t)]^2 \\ &= \lim_{N \rightarrow \infty} \frac{1}{N} \sum_{t=1}^N [bu(t-1) - u(t-1) - u(t-2) - w(t)]^2 \\ &= (b-1)^2 R_u(0) + R_u(0) + R_w(0) - 2(b-1)R_u(1) \\ &\quad - 2(b-1)R_{uw}(1) + 2R_{uw}(2), \end{aligned}$$

where

$$R_{ab}(\tau) = E[(a(t) - \bar{a})(b(t - \tau) - \bar{b})] = \lim_{N \rightarrow \infty} \frac{1}{N} \sum_{t=1}^{\infty} (a(t) - \bar{a})(b(t - \tau) - \bar{b})$$

Since  $w$  and  $u$  are independent, it is  $R_{uw}(\tau) = 0$  for all  $\tau$ , which gives

$$V(b) = (b-1)^2 R_u(0) + R_u(0) + \lambda_w - 2(b-1)R_u(1)$$

(a) If  $u$  is a white noise (of mean 0 and variance  $\lambda_u$ ) it is

$$R_u(\tau) = \begin{cases} \lambda_u & \text{if } \tau = 0 \\ 0 & \text{if } \tau \neq 0 \end{cases}$$

hence

$$\min_b V(b) = \min_b [(b-1)^2 \lambda_u + \lambda_u + \lambda_w]$$

yields  $b = 1$ .

(b)  $u(t) = (-1)^t$  gives  $R_u(0) = 1$ ,  $R_u(1) = -1$  etc. Hence

$$\min_b V(b) = \min_b [(b-1)^2 + 1 + \lambda_w + 2(b-1)] = \min_b [b^2 + \lambda_w]$$

gives  $b = 0$ .

(c) In the final case it is

$$\min_b V(b) = \min_b [(b-1)^2 + 1 + \lambda_w - 2(b-1)] = \min_b [(b-2)^2 + \lambda_w]$$

which is minimized for  $b = 2$ .

### 3. Exercise 3: System identification.

There is no sign of resonances, see the frequency function in Fig. 1, computed with SPA (M=100).

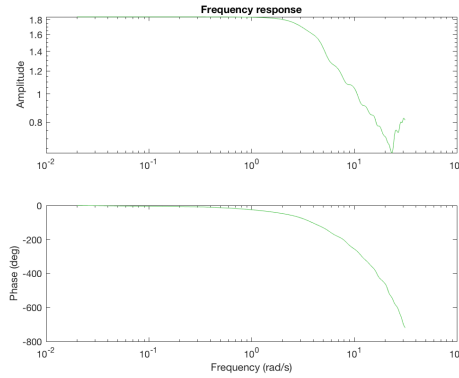


Figure 1: Frequency function.

The AXR order selection tool suggests a delay  $n_k = 3$ . Keeping this delay, a model that satisfies the constraint of max 3 poles is for instance the following OE(2,3,3):

oe233 =

Discrete-time OE model:  $y(t) = [B(z)/F(z)]u(t) + e(t)$

$$B(z) = 0.008766 (+/- 0.008597) z^{-3} + 0.9868 (+/- 0.01131) z^{-4}$$

$$F(z) = 1 - 0.4045 (+/- 0.01094) z^{-1} - 0.1036 (+/- 0.0141) z^{-2} - 0.001491 (+/- 0.008743) z^{-3}$$

Name: oe233

Sample time: 0.1 seconds

Parameterization:

Polynomial orders: nb=2 nf=3 nk=3

Number of free coefficients: 5

Use "polydata", "getpvec", "getcov" for parameters and their uncertainties.

Status:

Termination condition: Near (local) minimum, (norm(g) < tol).

Number of iterations: 2, Number of function evaluations: 5

Estimated using PEM on time domain data "mydatade".

Fit to estimation data: 93.77%

FPE: 0.01221, MSE: 0.01197

It provides a fit to validation data of 92.69%.

Parameter uncertainty is reasonably small, although for  $B(z)$  the first coefficient is of the same magnitude of the error (other models with max 3 poles seems to have a similar problem).

The model fit is shown in cyan in Fig. 2. Residuals are in Fig. 3 and are within

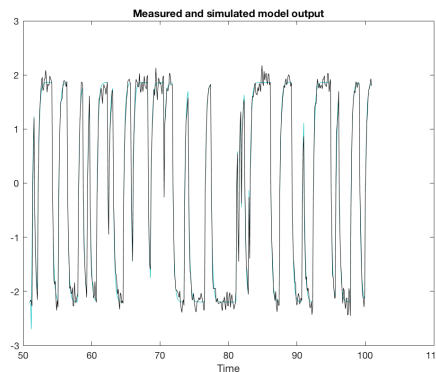


Figure 2: Model fit for OE(2,3,3).

ranges. Zeros/poles are in Fig. 5. The poles are all stable. Confidence interval are not completely disjoint, but again this appears due to the pole-order constraint. The frequency function is similar to the SPA frequency function, see Fig. 5, up to a difference in the DC gain.

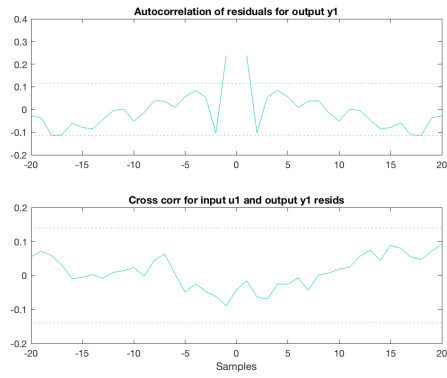


Figure 3: Residuals

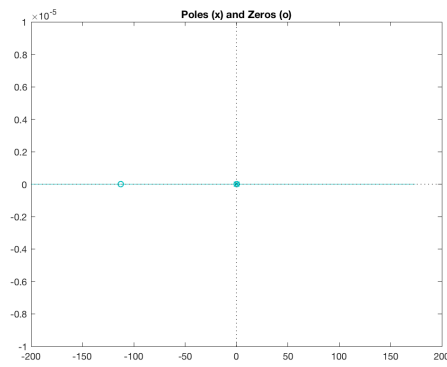


Figure 4: Zeros and poles

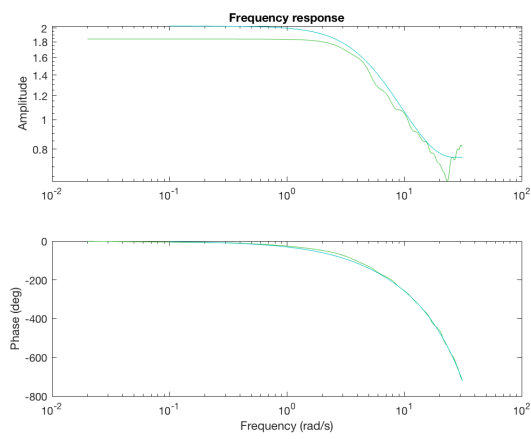


Figure 5: Frequency function of OE and SPA.

4. Exercise 4: bond graph

- (a) Denote  $(F_a, v_a)$  the (force, velocity) pair at the point  $a$  (top of the lever),  $(F_j, v_j)$  those at the junction point between the lever and the piston shaft (point  $j$  in the figure), and  $(F_b, v_b)$  the (force, velocity) pair at the point  $b$ . Comparing points  $a$  and  $j$ , the equation for the lever gives

$$\begin{aligned} \text{torque is the same: } F_a(L_1 + L_2) &= F_j L_2 \\ \text{angular velocity is the same: } \frac{v_a}{L_1 + L_2} &= \frac{v_j}{L_2} \end{aligned}$$

If  $(p_j, Q_j)$  and  $(p_b, Q_b)$  are the (pressure, flow) pairs at the pistons  $j$  and  $b$ , then flow incompressibility and no loss of energy in the liquid imply  $p_j = p_b$  and  $Q_j = Q_b$ . Using  $F_j = A_1 p_j$ ,  $F_b = A_2 p_b$  and the equations above for the lever, one gets

$$F_b = A_2 p_b = A_2 p_j = \frac{A_2}{A_1} F_j = \frac{A_2}{A_1} \frac{L_1 + L_2}{L_2} F_a$$

and similarly

$$v_b = \frac{A_1}{A_2} \frac{L_2}{L_1 + L_2} v_a$$

Hence  $n = \frac{A_2}{A_1} \frac{L_1 + L_2}{L_2}$  is the transformation factor of a single TF mapping from  $(F_a, v_a)$  to  $(F_b, v_b)$ .

- (b) The bond graph is given in Fig. 6 and its causality is conflict-free.

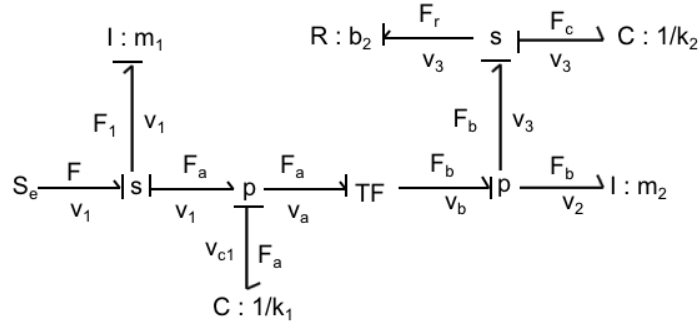


Figure 6: Exercise 4, bond graph

- (c) Let us choose  $F$  as input and as state variables the effort variables at the C-elements and the flow variables at the I-elements:

$$x = \begin{pmatrix} v_1 & F_a & F_c & v_2 \end{pmatrix}^T$$

With the choice of variable names shown in Fig. 6 one gets

$$\begin{aligned}
 \text{s:} \quad F &= F_1 + F_a \\
 \text{I:} \quad m_1 \dot{v}_1 &= F_1 \\
 \text{p:} \quad v_1 &= v_a + v_{c1} \\
 \text{C:} \quad 1/k_1 \dot{F}_a &= v_{c1} \\
 \text{TF:} \quad F_b &= nF_a \quad v_b = \frac{1}{n}v_a \\
 \text{p:} \quad v_b &= v_2 + v_3 \\
 \text{s:} \quad F_b &= F_r + F_c \\
 \text{I:} \quad m_2 \dot{v}_2 &= F_b \\
 \text{C:} \quad 1/k_2 \dot{F}_c &= v_3 \\
 \text{R:} \quad F_r &= b_2 v_3
 \end{aligned}$$

which yields the following ODEs

$$\begin{aligned}
 \dot{v}_1 &= \frac{1}{m_1} (F - F_a) \\
 \dot{F}_a &= k_1 \left( v_1 - n v_2 - \frac{n^2}{b_2} F_a + \frac{n}{b_2} F_c \right) \\
 \dot{F}_c &= \frac{k_2}{b_2} (n F_a - F_c) \\
 \dot{v}_2 &= \frac{n}{m_2} F_a
 \end{aligned}$$

## 5. Exercise 5.

- (a) The bond graph is in Fig 7 and it has conflict-free causality.  
 (b) Calling  $v$  the voltage at the generator and  $u_R / u_L / u_C$  and  $i_R / i_L / i_C$  the voltages and currents at R, L and C ( $u_C = u_L$ ), the state variables are  $u_C$  and  $i_L$ , and

$$\begin{aligned}
 \text{s:} \quad v &= u_R + u_C \\
 \text{R:} \quad u_R &= R i_R \\
 \text{p:} \quad i_R &= i_L + i_C \\
 \text{C:} \quad C \frac{du_C}{dt} &= i_C \\
 \text{I:} \quad L \frac{di_L}{dt} &= u_C
 \end{aligned}$$

which gives

$$\frac{d}{dt} \begin{bmatrix} u_C \\ i_L \end{bmatrix} = \begin{bmatrix} \frac{-1}{RC} & \frac{-1}{C} \\ \frac{1}{L} & 0 \end{bmatrix} \begin{bmatrix} u_C \\ i_L \end{bmatrix} + \begin{bmatrix} \frac{1}{RC} \\ 0 \end{bmatrix} v$$

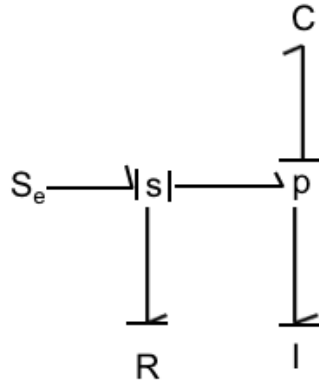


Figure 7: Exercise 5, bond graph for circuit with voltage generator

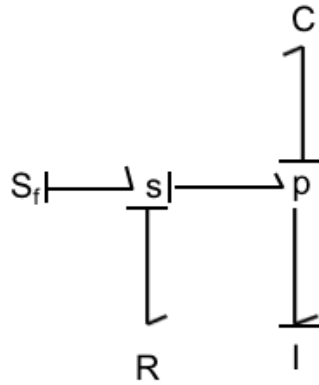


Figure 8: Exercise 5, bond graph for circuit with current generator

- (c) With a current generator, the new bond graph is in Fig 8 and it also has conflict-free causality. Notice the change in direction of the R-block causality stroke.
- (d) With the nonlinear model, for the first circuit the nonlinear ODEs are

$$\frac{du_C}{dt} = \frac{1}{C_1}i_C + \frac{1}{C_2}i_C^3 \quad (1)$$

$$\frac{di_L}{dt} = \frac{1}{L_1}u_C + \frac{1}{L_2}u_C^5 \quad (2)$$

The right hand side contains  $i_C$  which is not one of the two state variables, hence we need to add another equation to (1)-(2). In fact, to get rid of it, one must use  $i_C = i_R - i_L$  to express  $i_C$  as a function of  $u_C$ ,  $i_L$  and of the input  $v$ .  $i_R$  itself is not a state variable, therefore it must also be eliminated. However,  $u_R = v - u_C = R_1 i_R + R_2 i_R^7$  cannot be used to express  $i_R$  (and hence  $i_C$ ) in terms of  $u_C$  and  $v$ , hence one needs to differentiate this expression and pair it with the two ODEs

above.

$$\frac{du_R}{dt} = \frac{dv}{dt} - \frac{du_C}{dt} = (R_1 + 7R_2 i_R^6) \frac{di_R}{dt}$$

or, from  $i_C = i_R - i_L$ ,

$$\frac{di_C}{dt} = \frac{1}{(R_1 + 7R_2(i_C + i_L)^6)} \left( \dot{v} - \frac{1}{C_1} i_C + \frac{1}{C_2} i_C^3 \right) - \frac{1}{L_1} u_C + \frac{1}{L_2} u_C^5 \quad (3)$$

Notice that  $\frac{1}{(R_1 + 7R_2(i_C + i_L)^6)}$  is always well-posed when  $R_1 \neq 0$  (i.e., the “Jacobian”  $g_y$  in eq. (7.31) of the book is always invertible). Now (1)-(3) is a system of ODEs in  $u_C, i_L, i_C$  and input  $v, \dot{v}$ . We conclude that the differentiability index is 1.

If instead we consider the second circuit, the current  $i_R$  is now also the input of the system. Hence in (1)-(2) we can directly replace  $i_C$  with  $i_R - i_L$  (an input and a state variable). The differentiability index is 0 in this case.